

# Optimizing the Reliability and Efficiency of POD Cooling

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### Agenda

#### **Optimizing the Reliability and Efficiency of POD Cooling**

- 1. Row and perimeter cooling equipment for POD applications
- 2. Enhancing the reliability and efficiency of cooling equipment
  - 1. ASHRAE TC9.9: benefits of hotter return air
  - 2. ASHRAE 90.1: benefits of water side economizers
  - 3. Controls: benefits of matching cooling to IT load



### ASHRAE TC 9.9

#### **ASHRAE TC 9.9 Recommendations**

- <u>ASHRAE TC 9.9</u> Committee Thermal Guidelines for Data Processing Environments 2011 provides recommendations for temperature and humidity in the data center
- The standard is available for review
  - Simply Google ASHRAE TC 9.9 2011 and you'll find it

ASHRAE TC 9.9 2011 Thermal Guidelines for Data Processing Environments – Expanded Data Center Classes and Usage Guidance Whitepaper prepared by ASHRAE Technical Committee (TC) 9.9

Mission Critical Facilities, Technology Spaces, and Electronic Equipment

### ASHRAE TC 9.9



## Row Cooling Designs



Cooling with Cold Aisle Capturing



Cooling with Hot Aisle Capturing

### Indoor Cooling: Containment

#### **Applications / Configurations**

Raised floor with hot aisle containment and hot air return plenum



Slab floor with hot aisle chimney containment and hot air return plenum



### Perimeter Cooling Designs





Cooling with Cold Aisle Capturing

Cooling with Hot Aisle Capturing



Cooling with Cabinet Containment

## Row Cooling



### **Perimeter Cooling**



### ASHRAE TC 9.9

		6r6p60cv		
Entering Air DB (F)		75		
Entering Air WB (℉)		61.1		
Coil Leaving Air DB (F)		51.0		
Coil Leaving Air WB (°F)		50.5		
Gross Total Capacity (B	tuh)	513,800		
Gross Sensible Capacity	y (Btuh)	461,200		
Net Total Capacity (Btul	n)	493,800		
Net Sensible Capacity (	Btuh)	441,200		
Air flow (ACFM)		18,000		
External Static Pressure	e (in)	0.30		
Altitude (ft)		0		
Entering Fluid Temp. (F	-)	45		
Fluid Type		Water		
Percent Glycol(%)		0		
Fluid Flow (GPM)		105		
Leaving Fluid Temp.(%)		55		
Coil Fluid Pressure Drop	o (FT-H2O)	10.2		
Unit Fluid Pressure Drop	o (FT-H2O)	23.3		
Estimated Unit Pov	wer (kW)	5.3		
		1		

2 3

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#### 1. Base Coil Run

Raising entering (return) air temperatures by 20°F (11.2 °C) ...

- 2. Increases CRAH sensible capacity by 66%
- 3. Same or better sensible capacity with only **49% of the original fan power**
- 4. Allows higher chilled water temperature for **22%+ more efficient chiller operation**

### ASHRAE TC 9.9

#### What does this all mean?

- Higher return air temperature allows cooling equipment to operate more efficiently
  - Increasing return air temperature from 75° to 95°F = 66% increased efficiency
  - Improved efficiency leads to...
    - less energy, lower operating cost, and lower carbon footprint
  - Rack containment is a good way to increase return air temperature
- Higher return air temperature...
  - increases the hours of available free cooling, allowing further efficiency improvement

### **Presentation Contents**

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  - 2. ASHRAE 90.1: benefits of water side economizers
  - 3. Controls: benefits of matching cooling to IT load

### ASHRAE 90.1-2010

#### SHRAE 90.1–2010 Standard

- <u>ASHRAE 90.1-2010</u> Energy Efficiency for Buildings requires the use of air and water economizers in many locations
- The process cooling exemption is gone
  - Data Centers consume ~3% of total energy
- Water economizer must meet 100% of the expected load at:
  - Cooling towers: 40°F dry bulb / 35°F wet bulb
  - Dry coolers: 35°F dry bulb

#### ASHRAE 90.1-2010 Standard

- Huge impact on energy efficiency
- Leading-edge solutions
- Requires supporting equipment and controls
- New SCOP (sensible co-efficient of performance) requirements (replaces EER)



LOCATION, LOCATION, LOCATION Weather conditions are of primary concern when selecting an economizer type

### Air Side Economizers



- CRACs or CRAHs with an integrated mixing box and damper controls provides free cooling.
- Units attached to the top of a CRAC or CRAH in a vertical or horizontal position.
- Pre-filter and freeze protection damper options
- Mixes outside and return air depending on outside air conditions
- Full economizer controls provided by the CRAC or CRAH

### Side Economizers



#### **CRAC / CRAH with Economizer Mixing Box**

## Air Side Economizers - Cooling with Mixing Box

CRACs or CRAHs with an integrated mixing box and damper controls providing free cooling. Units attaches to the top of a CRAC or CRAH in a vertical or horizontal position.

#### Outside Air & Moisture Operation

Warmer / High Humid	ity Dampers close & the CRAC/CRAH reverts to traditional operation
Within Proper Range	Outside air is directly introduced through the dampers in the mixing box
Colder than Desired	Dampers mix outside air & return air to achieve desired temperature
<b>Below Freezing</b>	Warm return air mixes with outside air before the filter to prevent freezing

### Air Side Economizers Control



Air Side Economizer	Outside Temp.	OA Damper	Exhaust Damper	Return Damper	CRAC Fan	CRAC Compressor	Energy Use
FC (Free Cooling)	Cold	Variable	Variable	Variable	Variable	Off	Low
MIX (Compressor & Free Cooling)		Variable	Variable	Variable	Variable	Step	
DX (Compressor Cooling)	Hot	Closed	Closed	Open	Max	On	High

# How Controller manages this solution:

- Controls (variable 0-10V DC proportional control):
  - Outside Air Intake
  - Exhaust Damper or Fan
  - Return Air Damper
  - CRAC EC Fan
- Individual Control Selections
  - Fan
  - Outside Air Damper
  - DX Cooling

(Monitor contained aisle, outside air, supply air, return air, and a mix of outside and return air)

### Air Side Economizers Control





### Row Cooling

#### **Row Cooling – Chilled Water**

Valve

Filter ·

CW Coil



#### **Row Cooling – Direct Expansion**





#### CW heat rejection options:

- Chiller (air or water cooled) 1.
- Cooling Tower 2.

EC Fans



#### DX heat rejection options:

- Air cooled condenser (shown) 1.
- 2. Glycol cooled condenser
- 3. Water cooled cooling tower





#### **Dynamic Free Cooling**

Comprised of a variable fan speed dry cooler, variable speed pumps, and glycol cooled free cooling row-based cooling unit with both a DX and a glycol free cooling coil.

#### **Glycol Cooled DX**

Dynamic free cooling with variable speed dry cooler and pumps

#### **Outside Air** Operation

Warm Weather Months	Unit acts as traditional DX; dry cooler supplies glycol to unit condenser
In-between Months	Combination of glycol free cooling coil and one DX compressor (trim)
Cold Weather Months	Cooled glycol transferred to free cooling coil; compressor off





### Perimeter Cooling





#### **Traditional Free Cooling**

Comprised of a constant fan speed dry cooler, constant speed pumps, and glycol cooled free cooling CRACs consisting of both a DX and a glycol cooling coil.

Glycol Cooled DX Traditional free cooling with **constant** speed dry cooler and pumps

#### Outside Air Operation

Warm Weather Months	Unit acts as traditional DX; dry cooler supplies glycol to unit condenser
In-between Months	Combination of glycol free cooling coil and one DX compressor
Cold Weather Months	Cooled glycol transferred to free cooling coil; compressor off



#### **Dynamic Free Cooling**

Comprised of a variable fan speed dry cooler, variable speed pumps, and glycol cooled free cooling CRACs consisting of both a DX and a glycol cooling coil.

#### Outside Air Operation

Warm Weather Months	Jnit acts as traditional DX; dry cooler supplies glycol to unit condense					
In-between Months	Combination of glycol free cooling coil and one DX compressor					
Cold Weather Months	Cooled glycol transferred to free cooling coil; compressor off					
- fans on o	dry coolers and pumps operate at lowest possible speeds to					
supply required cooling, usir	ng the least amount of energy					
- as ambie	ent increases, fans on dry coolers and pumps increase					
speed to extend available free	ee cooling					



#### Evaporative Tower Free Cooling

Comprised of an evaporative cooling tower, constant speed pumps, and water cooled free cooling CRACs consisting of both a DX and a water cooling coil.

#### **Water Cooled DX** Free cooling with evaporative cooling tower and water filtration system

#### **Outside Air & Moisture Operation**

Warm Weather Months	Unit acts as traditional DX; cooling tower supplies unit condenser
In-between Months	Combination of water free cooling coil and one DX compressor
Cold Weather Months	Cooled water transferred to free cooling coil; compressor off



#### Free Cooling with Dual Chilled Water Coils

Comprised of an evaporative cooling tower, cooling tower pumps, chiller, chiller pumps, control valves, and chilled water cooled CRAHs.

#### **CW Units** Free cooling with evaporative cooling tower, water filtration system, and chiller

#### **Outside Air & Moisture Operation**

Warm/Humid Weather Months Chiller supplies CRAH

In-between Months Combination of water free cooling and partial chiller operation

Cold/Dry Weather Months Cooling tower directly supplies CRAH; chiller compressor off



#### Free Cooling with Chilled Water

Comprised of an evaporative cooling tower, cooling tower pumps, water cooled chiller, chiller pumps, control valves, and chilled water cooled CRAHs.

#### **CW Units**

Free cooling with evaporative cooling tower, water filtration system, and chiller

#### Outside Air & Moisture Operation

Warm Weather Months Cooling tower supplies chiller; chiller supplies CRAH

**Cold Weather Months** Cooling tower directly supplies CRAH; chiller compressor off

#### Water Side Economizer Control



### Warm Water Cooling

			6R12P60CV				
Water Tower Cooling Oper	Oversized un	it with Optimized C	oil	ooling Operation			
	at Non-Stand	ard Conditions					
Water Tower Chiller	Entering Air DB	(°F)	103	der			
	Entering Air WB	(°F)	67.5	CROH			
	Coil Leaving Air	DB (°F)	76.0	•			
	Coil Leaving Air	WB (°F)	58.1				
Chiller Pump	Gross Total Ca	pacity (Btuh)	318,500				
	Gross Sensible	Capacity (Btuh)	318,500				
	Net Total Capa	city (Btuh)	306,500	×			
	Net Sensible Ca	apacity (Btuh)	306,500	+			
- 5- ×	Air Flow (SCFM	)	10,800				
Water Tower Purro	External Static F	Pressure (in)	0.3				
	Fan Power (kW)	)	3.5				
	Altitude (ft)		0				
	Entering Fluid T	emp. (°F)	69				
	Fluid Type		Water				
	Percent Glycol(	%)	0				
	Fluid Flow (GPI	A)	25				
	Leaving Fluid T	emp.(°F)	94.5				
	Coil Fluid Press	ure Drop (FT-H2O)	2.3				
	Unit Fluid Press	sure Drop (FT-H2O)	6.5				
	Pump Power (k	W)	0.06				
	Total Power (kV	V)	3.6				
	Total Power to Standard Units	Usage to Match Net Sensible (kW)	7.8				

### ASHRAE 90.1-2010

#### What does this all mean?

- ASHRAE 90.1 2010 will have a huge impact on data center design
- Water-side economizer solutions
  - available for both row and perimeter cooling
  - optimized with rack containment
  - provide a huge improvement on cooling efficiency
    - Depending on your location and operating conditions, there is the potential to...
      - turn off compressors (CRAC or Chiller) for 80% of the year
      - 70% savings in overall energy usage

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  - 3. Controls: benefits of matching cooling to IT load

- Advanced Controls has a big impact on data center design
  - Cooling output dynamically matched to IT Load
    - Improves cooling operational reliability
    - Improves cooling efficiency and energy savings



#### Advanced Controls ties system together for maximum reliability and efficiency

- Inputs to the Network Gateway (monitor)
  - Wireless Temperature Sensors on the racks (rack entering air temp)
  - Pressure Transducers under the floor or in the cold aisle (pressure readings)
  - Wireless Control Modules on the CRAH's (supply or return air temperature)
- Outputs from the Network Gateway (control)
  - Controllers inside the CRAH's
- Controllers maximize reliability, free cooling, and energy savings
  - EC fans (speed controls)
  - Chilled Water Valves (GPM flow)
  - Variable Speed Pumps (GPM flow)
  - Variable Speed Dry Coolers (fan speed)







### **Economizers Control**

Figure 2. This illustration shows the data center recommended and allowed operating envelopes for ASHRAE classes A1 thru A4.



### Advanced Controls – Active Redundancy

<b>Examples</b>			Airflow:
1.) 3 x CRAH's @ 75° F / 50% EAT with 45° F water operating with 1/3 standby capacity			2 x 17,000 cfm = 34,000 CFM
	Standby	Two Units @ 370,000 Btu/Hr each = total 740,000 Btu/Hr	Fan energy consumption 2 x 9.1 = <b>18.2 kW</b>
2.) 3 x CRAH's @ 75° F / 50% EAT with 45° F water each operating at the same time			<b>Airflow:</b> 3 x 11,333 cfm = 34,000 cfm
	Three Uni = to	ts @ 275,000 Btu/Hr each tal 825,000 Btu/Hr	Fan energy consumption 3 x 2.69 = <b>8.07 kW</b>

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Operating all CRAH's at a lower fan speed provides better redundancy and energy savings

Delta of 10.13 kW / year = **\$8,874 energy savings / year** (@ \$0.10 / kWh)

## **Cooling Complexity**

Advanced Controls measures and models changing heating and cooling conditions during commissioning, additions, and changes



#### **System Monitoring**

- Detailed reporting of data center cooling operations
- Trending, performance, and compliance
- Displays rack temperatures and temp. distribution
- Identifies areas of unused capacity

#### **System Control**

- Real-time cooling capacity management
- Cooling output dynamically matched to IT Load
  - shuts down unnecessary CRAC units
  - Optimizes CRAH fan speed and water flow
  - Cooling output dynamically matched to IT Load

# Advanced Controls measures, models, and manages data center cooling infrastructure



#### **Increased Reliability**

- Customizable alarms
- Instant, automated response to cooling problems

#### **Efficient Operations**

- Hot spot prevention
- Up to 50% reduction in equipment cycling
  - Increased equipment life

#### **Energy Savings**

- Continuous PUE optimization
- Up to 60% cooling energy savings



Key Control Capabilities	Benefit
Independent Control	Efficiency of EC fans and CW valve (or compressor) can be maximized
Dew Point Control	Temp and RH examined and humidification/dehumidification modes optimized
Static Pressure Control	Provides even airflow on demand at any room position and precise control for containment
Unit Networking	Capable of BMS interface or private local area network (pLAN)
Redundancy	Controller defaults to set points if a sensor fails or if BMS control signal fails; EC fans ramp up if one fails

### **Ultrasonic Humidification**



#### Ultrasonic

Humidification for mission critical applications and any environment where clean, efficient and tight humidity control is required.

**93%** energy savings over infrared humidifiers.



### Humidification Improves Efficiency



#### Is Ultrasonic right for your data center?

#### Key Environmental Criteria:

- Moisture content below the ASHRAE minimum recommendation level will lead to electrostatic discharge (ESD) failures of IT equipment:
  - Compromise equipment reliability
  - Costly system downtime
  - Costly equipment replacement

#### Key Site Qualification Criteria:

- Review the data center environment conditions
  - Location (dry verses humid environment)
  - Infiltration of outside air (vapor barrier)
  - Design day (worst case dry day < 41.9°F DP)

### Humidification Improves Efficiency

Feature	Infrared Humidifier	Ultrasonic
Technology	Infrared	Ultrasonics
Capacity (lbs./hr.)	22	18
Energy Consumption (Watts)	9,600	585
Water Requirements	Non-conditioned city water	Demineralized water using RO and DI
Maintenance Requirements	<sup>1</sup> / <sub>2</sub> hour to remove mineral buildup in pan	No cleaning of ultrasonic required

#### Humidification Energy Comparison

Humidification Technology	Number of Humidifiers Required		Watts Consume d		Annual Hours Of Operation		KW/H per year		Cost Per KWH		Annual Cost of Electricity	Annual Savings
Infrared	98	х	9,600	х	3,725	х	3,504,480	х	\$0.11	х	\$385,493	
Ultrasonic	120	x	585	х	3,725	х	261,495	х	\$0.11	х	\$28,764	\$356,728

#### **Free Cooling Energy Savings**

	Adiabatic cooling provided (BTU/h)	Total KW/H per year*	Cost Per KWH	Annual Cost of Electricity	Humidification and Free Cooling Energy
Ultrasonic	2,095,200	799,973 x	\$0.11	\$87,997	Savings
		*IAAU II A			\$444.725

#### Combined

### **Ultrasonic Humidifier in Space**



### **Economizers Control**

Figure 2. This illustration shows the data center recommended and allowed operating envelopes for ASHRAE classes A1 thru A4.



### Outdoor Cooling: Modular Container



Outdoor modular container system is designed for conditioning the air in a containerized computer room (POD).

#### **Features and Benefits**

- Controller ensures precise operation
  - ✓ Separate control cabinet per integral cooling unit
- ETL & cETL Listed
  - ✓ Compliance with UL 1995 Standards

### Outdoor Cooling: Modular Container

#### **Design Principle and Operation**



# Modular Container Solution: Air-Side Economizers



Allows modular build out of containerized data centers

Owners can increase data center capacity as their demand increases

### Air Side Economizers



#### Air Side Economizers - Integrated into Air Handler Unit

Air Handler with an integrated mixing box and damper controls provides free cooling. Can be mounted on top of a building or ducted to the side of a building.

#### **Outside Air & Moisture Operation**

Warmer / High Humidity	Dampers close & the CRAC/CRAH revert to traditional operation
Within Proper Range	Outside air is directly introduced through the dampers in the mixing box
Colder than Desired	Dampers mix outside air & return air to achieve desired temperature

### Air Side Economizers



Outside Air to Air Handler 68.0° F db 61.1° F wb 57.0° F dp

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#### Air Side Economizer Integrated into Container or Air Handler

Air side economizer with direct adiabatic cooling provides the lowest possible energy consumption

Supply Air to					
Data Center					
69.1°Fdb					
61.5°F wb					
57.0°Fdp					

iter db wb dp	68℉ Outside Air 105℉ Return Air 69.1℉ Supply Air	DX Cooling Only	Air-Side Economizer and DX for Trim	Air-Side Economizer, Direct Adiabatic Cooling, and DX for Trim
	Power Required	123.5 kW	83.4 kW	67.7 kW
	Energy Savings	0 kW	40.1 kW (32%)	55.8 kW (45%)







### **Direct Adiabatic Cooling - Operation**

- Direct adiabatic cooling systems introduce water for evaporation directly into the supply airstream
- Direct adiabatic cooling options:
  - 1. Using Ultrasonic Humidifier
  - 2. Using an adiabatic pad
  - 3. Using a high pressure spray system
- Integrated controls monitor conditions to prevent over-humidifying
- Direct adiabatic cooling utilized with an air-side economizer provides the lowest possible energy consumption.



### Non-Mechanical Cooling Solution



**Outdoor Direct Adiabatic Cooling Unit** 

- Evaporative Pad
  - non-recirculating water (no pump, no filter)
  - minimal water waste (precise absorption control)
- Air-Side Economizer with mixing dampers

### **Cooling Solutions**



### **Indirect Adiabatic Cooling**



#### **Energy Efficiency Example**

- Starting DP temp 86° F and WB temp 66° F (20° delta)
- Possible temperature drop of 14° F
- The > the difference between the WB and DB temp's, the > the achievable temperature reduction

#### **Indirect Adiabatic Module**

- 1. Return air flows through the heat exchanger
- 2. Outside air flows through the heat exchanger
- 3. Liquid water is introduced to the heat exchanger to reduce the temperature of the outside air
  - 1. Evaporation takes place on the heat exchanger
  - 2. Water evaporates into the outside air stream
- 4. Return air is cooled through the heat exchanger
  - 1. No mixing of outside air and return air

### Air Handler with Indirect Adiabatic



- When outside air conditions are favorable, the indirect evaporative system is active
- On cooler days, the system can be used without evaporation as an air-to-air heat exchanger
- When outside air temperature or humidity level are not favorable for free cooling, then
  - Turn off water to heat exchanger
  - Turn off heat exchanger exhaust fans to prevent heating of the return air

### Air Handler Solution - Traditional



- Open data center designs allow hot return air to mix with cold supply air
- This mixed air surrounds the server racks, and is then used as the control point
- This results in lower return air temperature and requires lower supply air temperatures
- Latent cooling is often a result of lower supply/return temperatures

### Air Handler Solution



- Isolating cold supply air from hot return air prevents mixing of the air streams and short-cycling
- All cold supply air is used for cooling; all hot air goes back to the AHU return (or is exhausted)
- Supply air temperature can be used as the control point (no mixing of hot and cold air)
- Higher supply temperatures and higher return temperatures can be utilized
- The cooling equipment operates at higher efficiency with greater capacity

### **Pre-Engineered AHU's**

Design to optimize to be shorter and narrower Allows for international shipping inside a shipping container



Integrated Condensing Unit / High Static Supply Fans / Adiabatic Cooling / Integrated Exhaust



### Air Handler Development

has over 20 years of experience designing and building air handling equipment has the ability to meet customer application-specific requirements





### Utility Rebates Further Improve ROI

#### **Utility Rebates Further Improve ROI**

- Many utility companies are running out of capacity
- Most utility companies are offering incentive programs for:
  - Retrofitting existing cooling equipment
  - Replacing cooling equipment
  - Installing new energy efficient cooling equipment



## Key Takeaways

#### **Optimizes the Reliability and Efficiency of POD Cooling**

- 1. State-of-the-art row and perimeter cooling equipment for POD applications
- 2. Enhances reliability & efficiency of cooling equipment
  - 1. ASHRAE TC9.9: hotter return air increases efficiency
  - 2. ASHRAE 90.1: water side economizers increase efficiency
  - 3. Controls: matching cooling to IT load increases efficiency

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